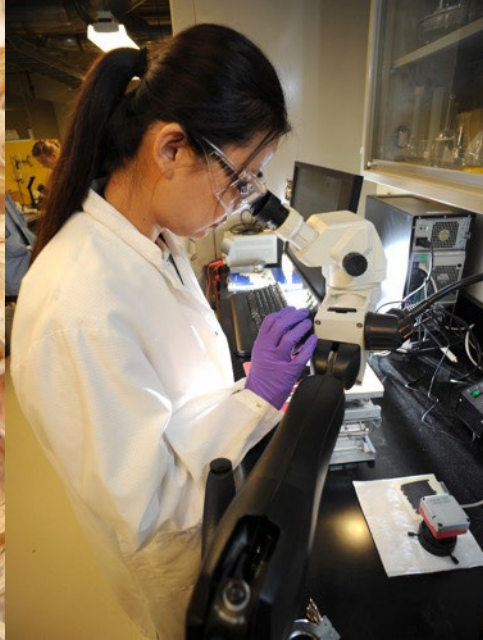


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3D Heat Conduction for Helical Cruciform Fuel in RELAP5-3D

Battelle Energy Alliance manages INL for the
U.S. Department of Energy's Office of Nuclear Energy



Idaho National Laboratory

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Presentation Agenda

1. Helical Cruciform Fuel Rod
2. Heat Conduction Equation
3. 1D heat conduction in R5-3D
4. 3D heat conduction complications
5. Aspects of Finite Volume Method
6. Conclusion

Note: These are interesting preliminaries leading to final decisions on modeling.

Adding Helical Cruciform Rod (HCR) to R5-3D Heat Conduction

- Current Geometries / Shapes: Rectangular, cylindrical, spherical
- New Shape: **4-lobe Helical Cruciform** – many complications



Figure 1. Length for 1 360 twist of Helical Cruciform Rod

Figure 2. Cross section of 4 lobe Helical Cruciform Rod

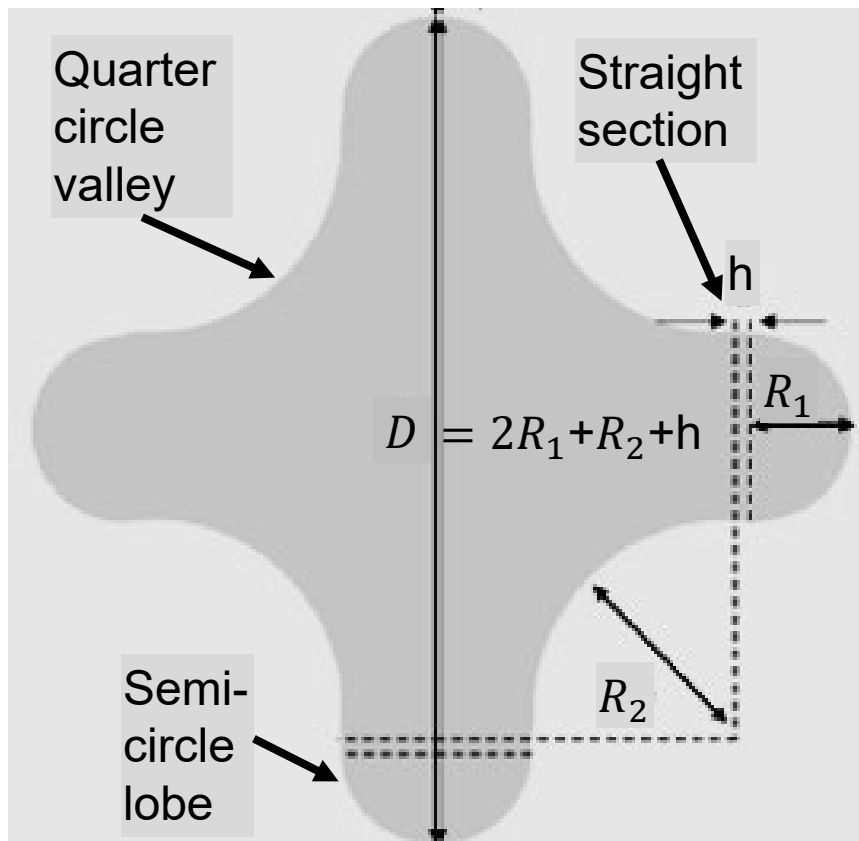
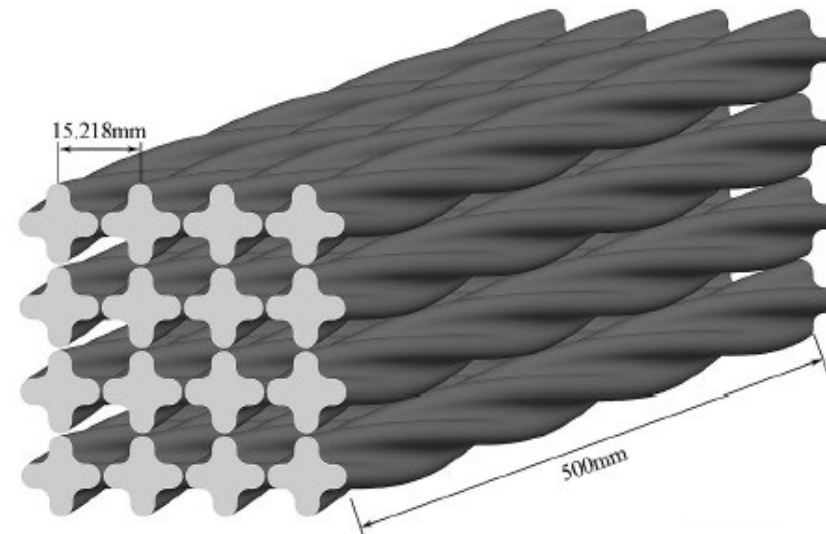


Figure 3. Stack of HCF Rods

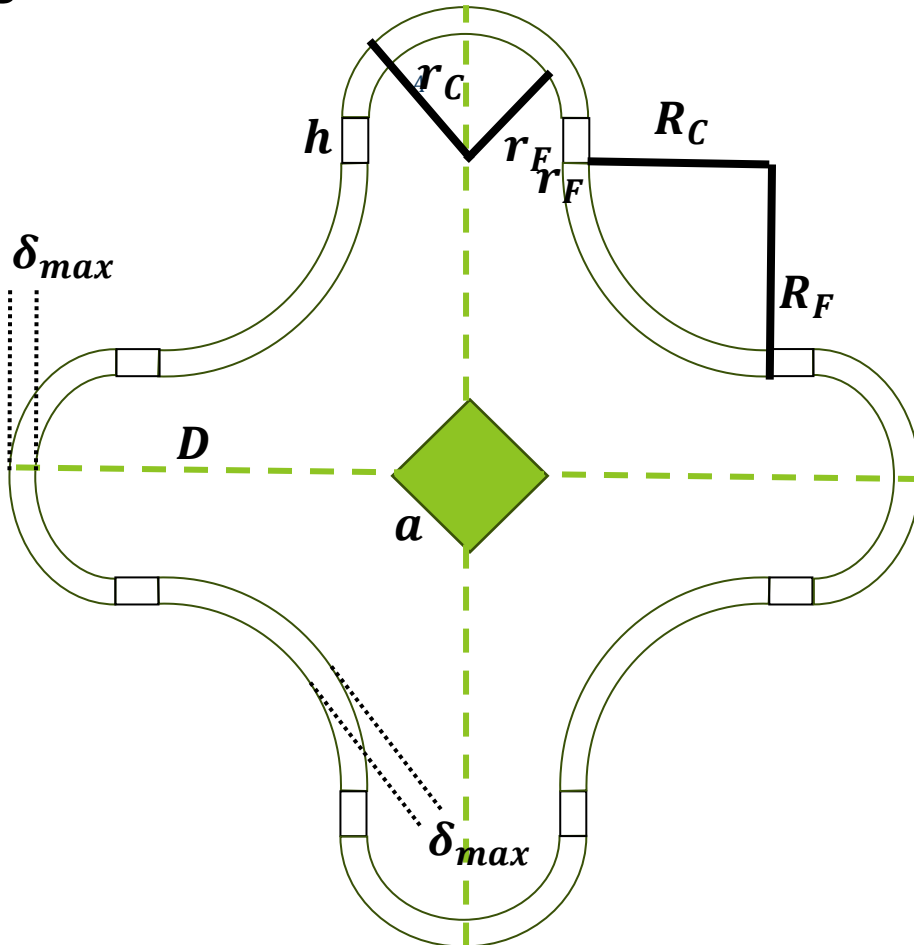


The HCR touch, have crossflow, and Cause turbulence, Need correlations.

Helical Cruciform Rod (HCR) to R5-3D Heat Conduction

- 4-lobe Helical Cruciform FUEL ROD has many complications for modeling

Figure 4. Detailed HCF Rods cross section



- Fuel shown in purple
- Cladding in pink has varying depths
 - δ_{min} in the valley
 - δ_{max} in the lobe
 - δ_{avg} in between.
 - Notation: R for valley, r for lobe, C for clad, F for fuel, a for displacer side length
- Center non-square “displacer” holds a nuclear “poison” has length **a** on each side.
- **INPUT:** User must specify **21 characteristic values** on the 800-999 cards

Some projected improvements with possible HCR Fuel:

- The fuel may have high thermal conductivity and low swelling.
- The cladding has variable thickness to increase protection at the lobe tips.
- Design allows existing and new reactors to operate at lower temperatures while extracting more heat from the fuel core
 - Thus delivering **greater electricity output**.
- Greater fuel surface area than cylindrical fuel rods could produce more margin to fuel failure.
- The displacer contains *burnable poison* alloys for neutronics control.

Some projected improvements with HCR include:

- Metallurgical bonding of the displacer, fuel, and cladding could:
 - Improve fuel rod integrity and thermal conductivity
 - Eliminate a source of fission product release in the event of a bonded barrier breach
- The operating temperature can be much cooler than that of standard nuclear fuel.
- Reduced fuel operating temperature should give:
 - More active cooling time and
 - Better structural integrity

Heat Conduction Equation



- $$\frac{\partial[(\rho C_p)(\bar{x}, T) \cdot T(\bar{x}, t)]}{\partial t} = \bar{\nabla} \cdot k \bar{\nabla} T + S$$
- Integrate and apply divergence thm
- $$\iiint_V (\rho C_p)(\bar{x}, T) \frac{\partial T(\bar{x}, t)}{\partial t} dV = \iiint_V S(\bar{x}, T) dV - \iint_S k(\bar{x}, T) \bar{\nabla} T(\bar{x}, t) \cdot d\bar{s}.$$
- With 1D finite Differences, can model:
 - Rectangular
 - Cylindrical
 - spherical shapes
 - Details in Vol. 1 – Theory Manual

ρC_p	= volumetric heat capacity	$(J/[m^3 K])$
k	= thermal conductivity	$(W/[mK])$
S	= internal volumetric heat source	(W/m^3)
T	= temperature	(K)
V	= volume	(m^3)
$d\bar{s}$	= surface normal area vector	(m^2)
t	= time	(s)
$\bar{x}$	= position	$(m).$

1D Heat Conduction

The code uses 1D finite differences for these shapes

- For rectangular, mesh points sit on x-axis going left to right
- Temp. at x_m represents avg. across slab in y- & z- directions
- Properties k and ρC_p are measured at grid points x_k

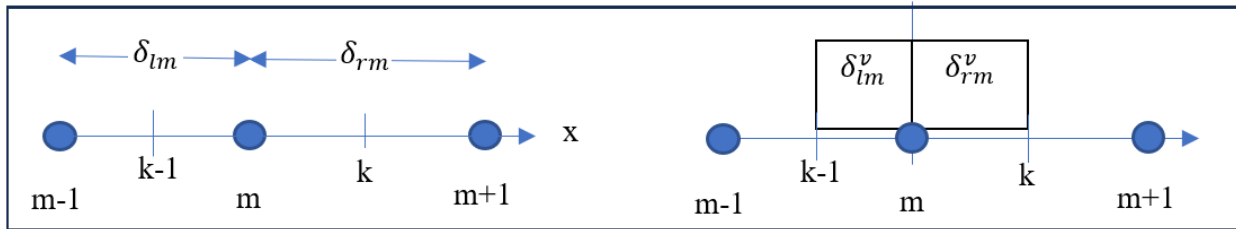


Figure 5. 1D rectangle mesh & grid point locations

- For cylindrical, pts. sit on a ray going outward from the center
- Temp represents integral across circular shell at radius x_m .

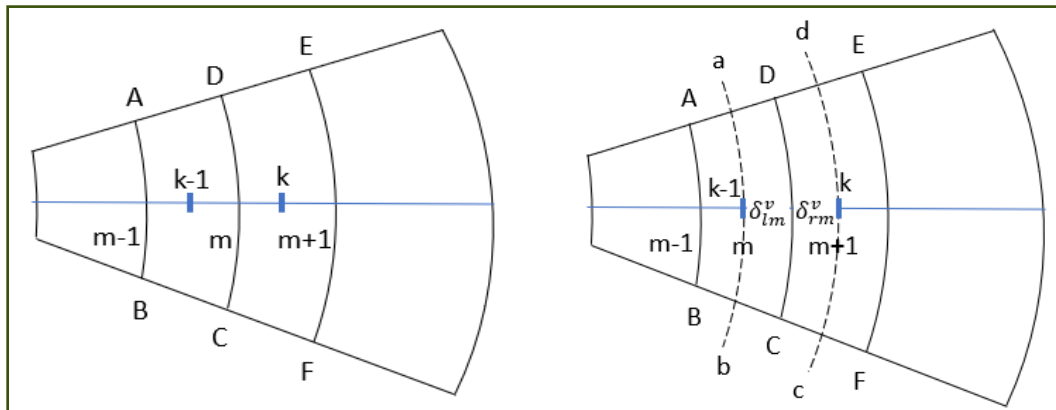


Figure 6. 1D cylindrical mesh & grid point locations

The undulating exterior cross-section and twisting shape preclude 1D modeling for the Helical Cruciform Fuel Rod.

A 3D Heat Conduction Discretization without Displacer

- One Finite Volume 3D discretization has
 - A axial levels (Fig. 7)
 - R heat structure rays / level (Fig. 8)
 - M mesh points (Fig. 8)
- Mesh points on all rays are scaled multiples of the ray 1 mesh points.
$$\|x_{a,r,m} - C\| = \frac{L_r}{L_1} \|x_{a,1,m} - C\|$$
 - L_1 =length ray 1
 - L_r =length ray r
 - C = center
 - $x_{a,r,m}$ = mth mesh point from center on axial level a, ray r

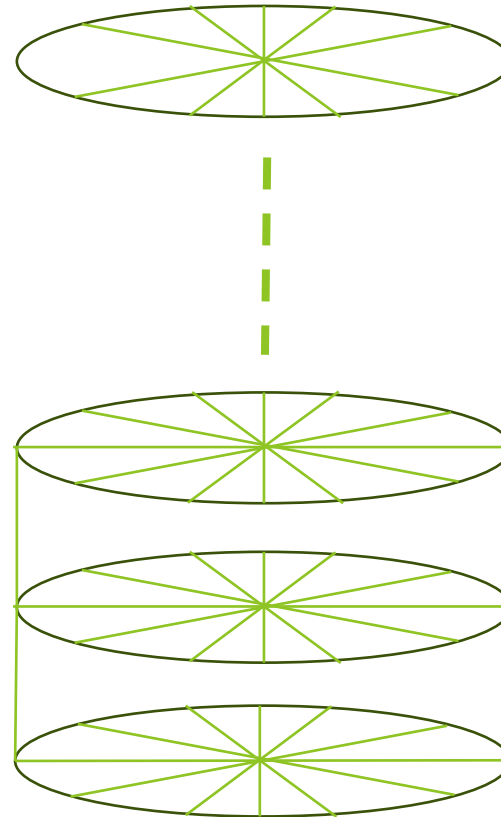


Figure 7. Axial levels in a 3D cylinder

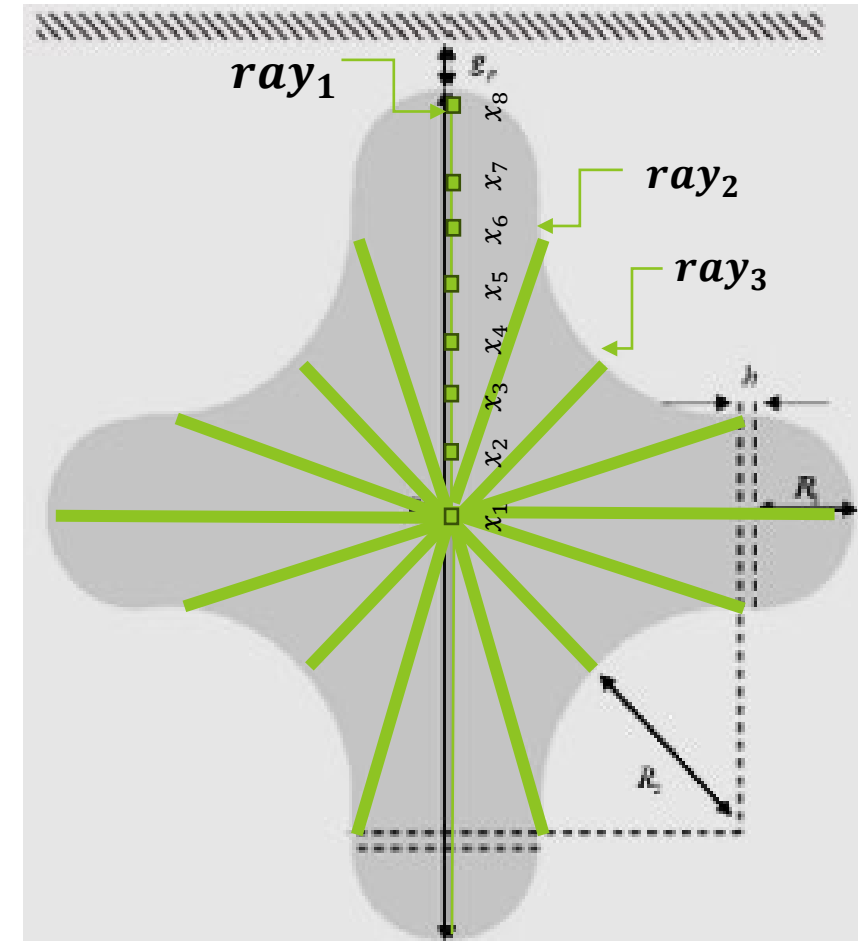


Figure 8. Heat structures along rays.

A 3D Heat Conduction Discretization without Displacer

- Angle between all pairs of rays, $\theta = 360/N_{ray}$;
 - For symmetry, the number of rays, N_{ray} , must be a multiple of 4.
- Simple configuration has complications
- Figure 5 shows the three categories of Heat Structure rays that occur, based on angle θ
 - (1) Ray intersects a lobe
 - (2) Ray intersects a valley
 - (3) Ray intersects a straight section (A.K.A. h-part)
 - (3a) Special case: Intersects junction with valley
 - (3b) Special case: Intersects junction with the lobe.
- The **next 2 slides** give **formulas for ray lengths** in each category needed for locating the mesh points

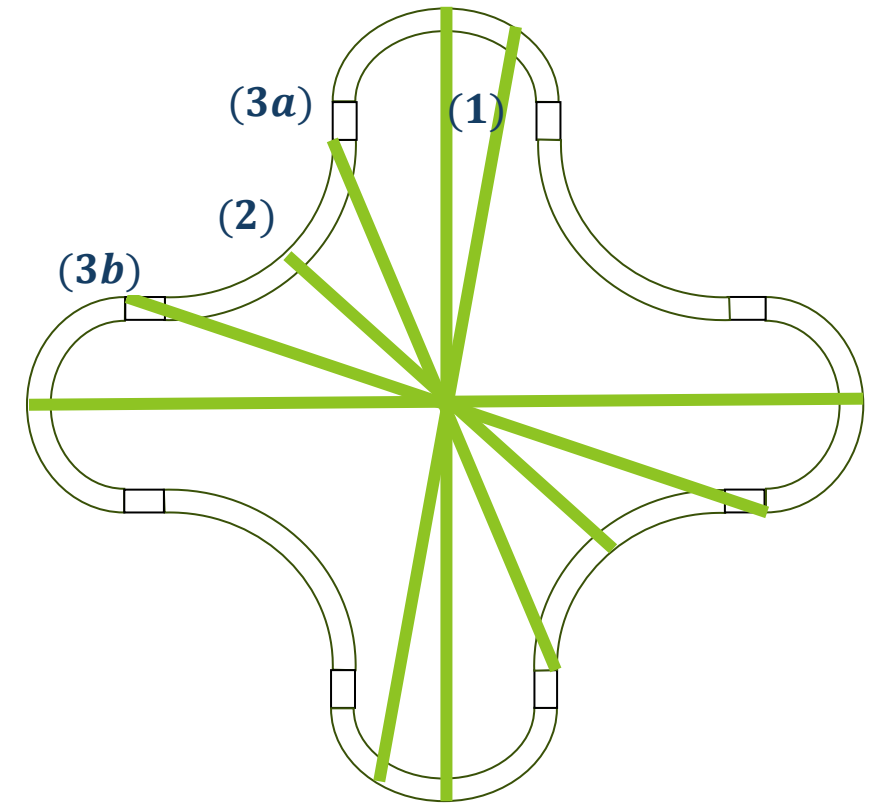


Figure 9. The 3 categories of heat structure rays

A 3D Heat Conduction Discretization without Displacer

- The HCRF circumscribed diameter is
- $D = 2(R_2 + 2R_1 + h) = 2(r_F + 2R_F + h + \delta_{max})$
- For the circle circumscribed about the outer edges, half the diameter D is the length of ray 1.
- $L_{ray1} = r = [R_2 + 2R_1 + h] = r_F + 2R_F + h + \delta_{max}$
- For Category (1) rays, the length is
- $L_{ray(k\theta)} = [R_1 + R_1 \sin(\angle DAB) + R_2 + h] \sec(k\theta)$
where

$$\alpha = \angle DAB = 90 - k\theta - \sin^{-1} \left(\frac{[R_1 + R_2 + h] \sin(k\theta)}{R_1} \right)$$

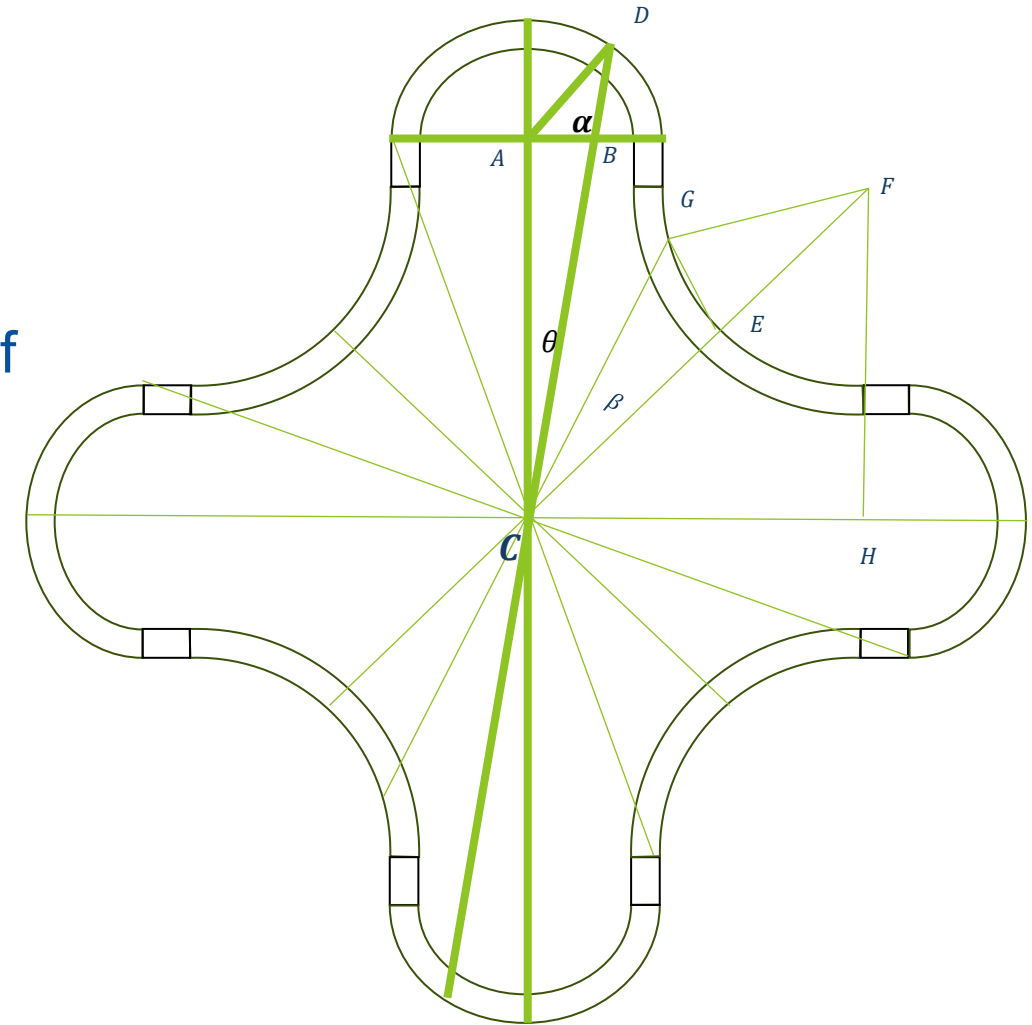


Figure 10. Illustrating Category (1) calculation

A 3D Heat Conduction Discretization without Displacer

- For Category (2), $L_{r(k\theta)} = \frac{-B \pm \sqrt{B^2 - 4AC}}{2A}$
 - where
 - $A = 1$,
 - $B = 2\sqrt{2}[R_1 + R_2] \cos(k\theta)$,
 - $C = 2[R_1 + R_2]^2 - R_2^2$
- For Category (3), $L_{r(k\theta)} = R_1 |\csc(k\theta)|$
- **This gives free 3D cylindrical coordinate form**
 - **Allows Verification testing against analytical solutions**
- NOTE: These rays start at the Center
- *Must adjust for the center displacer square*

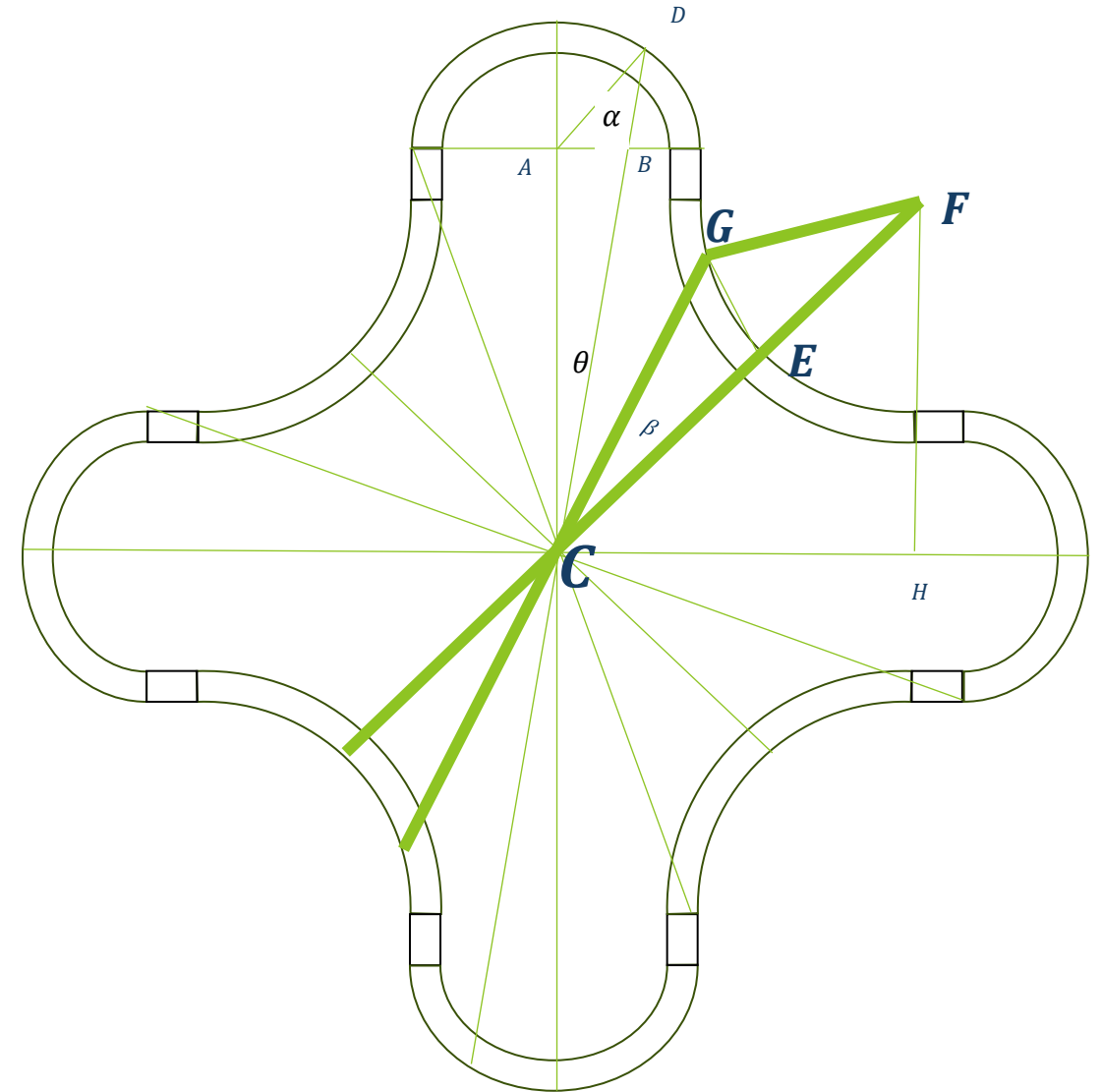


Figure 11. Illustrating Category (2) heat structure rays

A 3D Heat Conduction Discretization with Displacer

- To account for the displacer “square”
- Calculate, $L_{r(k\theta)}^a$, distance from center to ray $r(k\theta)$ intersection w/ displacer’s edge.
 - Can treat displacer it as a square –LTBR.
- $L'_{r(k\theta)} = L_{r(k\theta)} - L_{r(k\theta)}^a$
 - where
$$L_{r(k\theta)}^a \frac{a}{\sqrt{2}} \sqrt{1 + \cos^2(k\theta - 45) - \cos(k\theta - 45) \sin(k\theta - 45)}$$
- This applies to all three categories of rays.

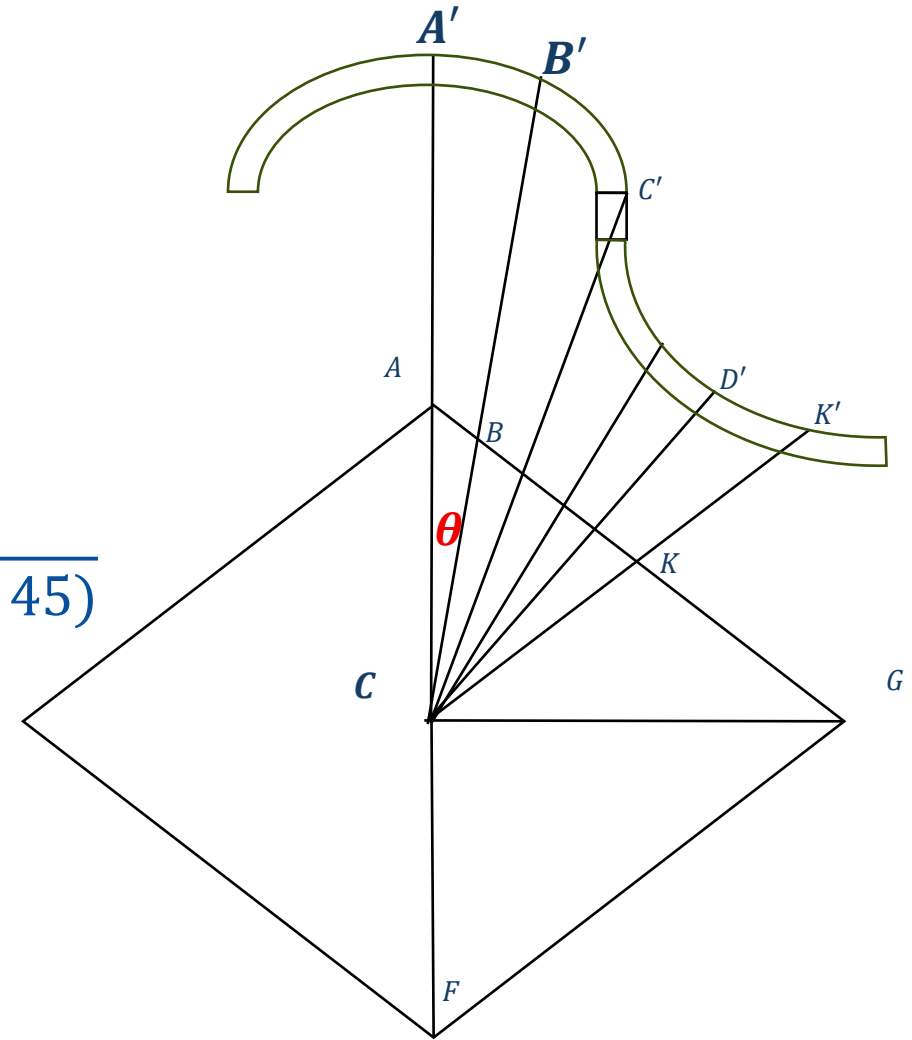


Figure 12. Illustrating Category (2) heat structure rays

3D HC Discretization Finite Volume

- The formula for grid points to produce faces whose normal vectors are parallel with the r-axis is:
 - $x_{a-\frac{1}{2},r-\frac{1}{2},m+\frac{1}{2}} = x_{a-\frac{1}{2},r+\frac{1}{2},m+\frac{1}{2}} = [x_{a,r,m+\frac{1}{2}}] \tan \frac{\theta}{2}$
 - $x_{a-\frac{1}{2},r-\frac{1}{2},m-\frac{1}{2}} = x_{a-\frac{1}{2},r+\frac{1}{2},m-\frac{1}{2}} = [x_{a,r,m-\frac{1}{2}}] \tan \frac{\theta}{2}$
 - $x_{a+\frac{1}{2},r-\frac{1}{2},m+\frac{1}{2}} = x_{a+\frac{1}{2},r+\frac{1}{2},m+\frac{1}{2}} = [x_{a,r,m+\frac{1}{2}}] \tan \frac{\theta}{2}$
 - $x_{a+\frac{1}{2},r-\frac{1}{2},m-\frac{1}{2}} = x_{a+\frac{1}{2},r+\frac{1}{2},m-\frac{1}{2}} = [x_{a,r,m-\frac{1}{2}}] \tan \frac{\theta}{2}$
- However, the entire HCR twists helically which forces the corner edges to twist into curves.

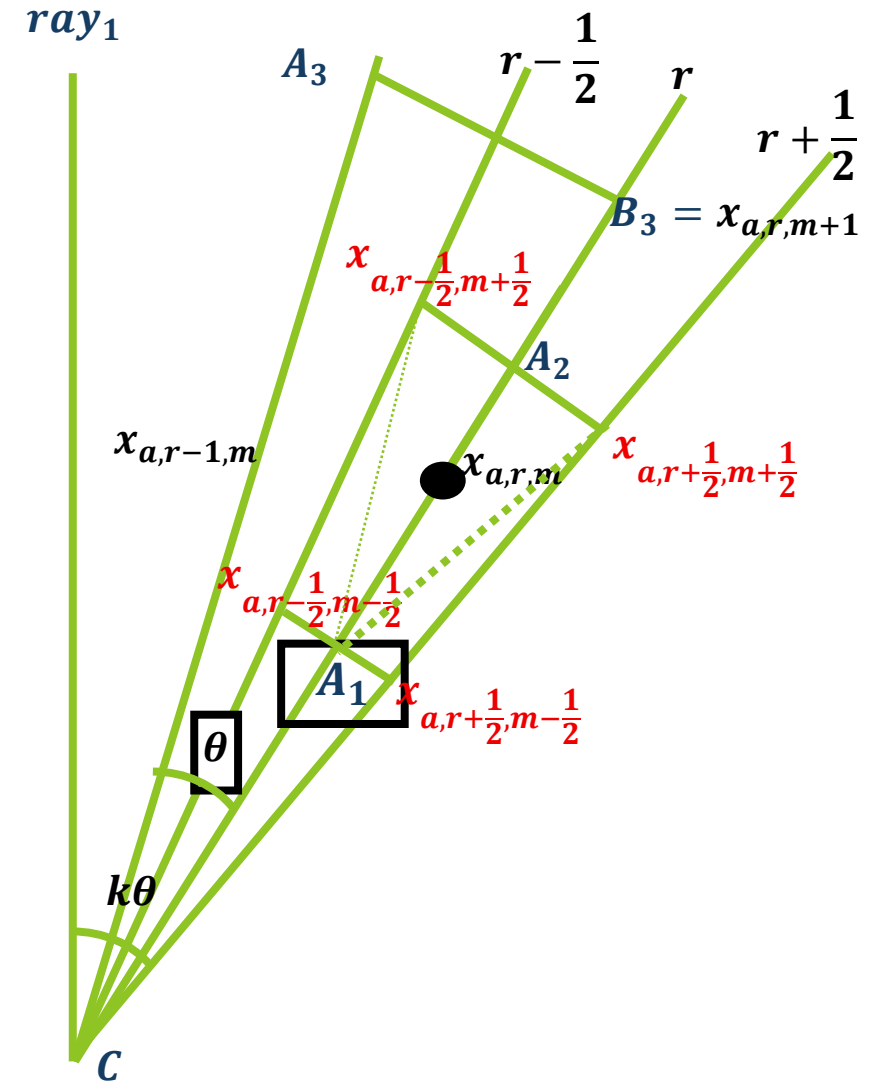
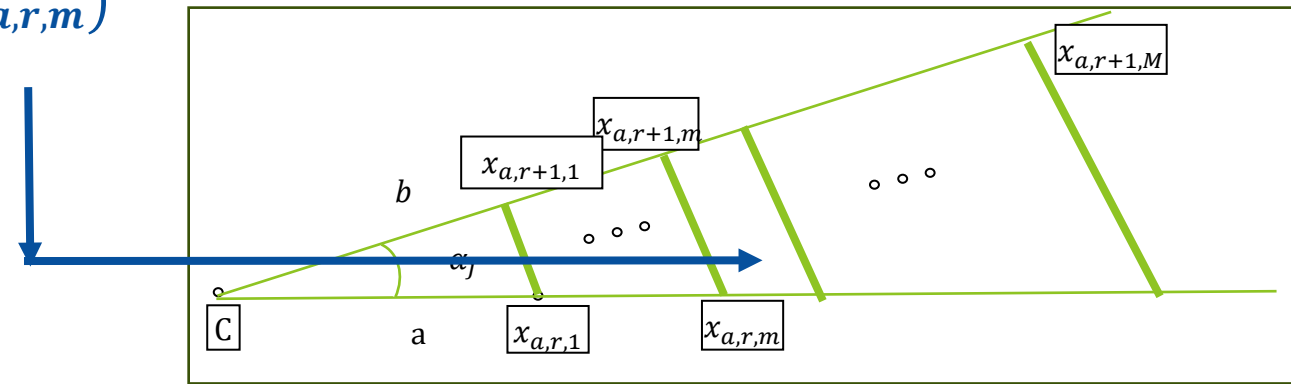


Figure 14. Arranging points so the face is normal to the r-direction.

The Volume of the Finite Volume

- In rectangular coordinates is $(x = r \cos(\alpha), y = r \sin(\alpha), z)$.
- The arclength of this path is the integral of the derivative w.r.t. axial height, $L_{z,a}$, from one axial level to the next.
- $\alpha_{twist,a} = 360^\circ \left(\frac{L_{z,a}}{H} \right)$, H = pitch = length to make one 360° twist.
- $Path_{twist,a} = \left(r \cos \left(\frac{360^\circ z}{H} \right), r \sin \left(\frac{360^\circ z}{H} \right), z \right), 0 < z < L_{z,a}$
- $L_{twist,a,r} = \int_0^{L_{z,a}} \left\| \frac{dPath_{twist,a}}{dz} \right\| dz = \int_0^{L_{z,a}} \sqrt{r^2 \sin^2 \left(\frac{360^\circ z}{H} \right) + \cos^2 \left(\frac{360^\circ z}{H} \right) + 1^2} dz$
- $L_{twist,a,r} = \int_0^{L_{z,a}} \sqrt{r^2 + 1} dz = (r^2 + 1)^{0.5} z \Big|_0^{L_{z,a}} = L_{z,a} \sqrt{r^2 + 1} = \text{arclength}$
- $V_{grid}(x_{a+\frac{1}{2},r+\frac{1}{2},m+\frac{1}{2}}) = A_g(x_{a,r,m}) L_{twist,a,r}$
- Subtracting consecutive triangles produces $A_g(x_{a,r,m})$

Figure 15. Illustrating Category (2) heat structure rays





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1. The central “square” has sides that are NOT normal to the r-direction.
 2. The surface of the HCR is also NOT normal to the r-direction.
 3. The surface can have two different formulas
 1. Part circular loop section and part straight section
 2. Part circular valley section and part straight section
- This creates many special cases for programming
- Introduces programming complexities
 - Creates Verification and Validation issues

Finite Element Method

- Finite Volume Method for 3D Heat Conduction has numerous complications.
 - With skewed faces do not get first order accuracy.
- Finite elements simplifies many of these.
 - The elements can be standard tetrahedra and polyhedra
 - No need for special curved edges
 - They can match the interior displacer “square” without issue
 - It also simplifies nicely to cylindrical geometry, so that’s gotten for free also.
- First order accuracy is obtained even if the HCR surface is only approximated.
- Many grid generators are available for the Finite Element Method.
- It requires a whole new and modern input processing for the grid
 - But this will open new possibilities for modeling other physical phenomena.
- Details for FEM modeling are just getting underway.
- Decision after comparison of complications each introduces.

Conclusion

- Finite Volume Method for 3D Heat Conduction is worked out.
- It produces an elegant simplification to 3D Cylindrical Coordinate form for cylindrical fuel rods.
- Unfortunately, there are numerous problems with the boundaries
 - The central square does not match the natural coordinate system.
 - The edges have numerous complications requiring adjustment by a Jacobian matrix of transformation.
 - Finite volume methods do not place temperature on the actual boundary, so the issues with losing first order accuracy may not apply

Conclusion

- The Finite Element Method will be worked out.
 - It naturally matches up boundaries near the central “square”
 - It can still be first order accurate even if the exterior boundaries are not perfect.
 - Use of curved edges on the exterior boundary will yield 2nd order accuracy.
- The two methods will be compared on paper and a decision made.